

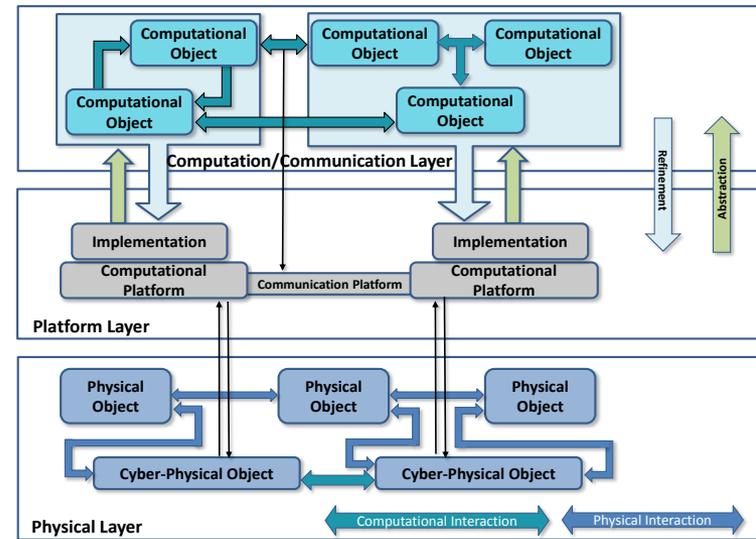


# **Model-Based Design and Verification of Automotive CPS**

**Xenofon Koutsoukos**



# System Integration



- Implemented components are connected and system-level properties are verified/tested
  - **High risk** – many fundamental problems surface during system integration
  - **Ad-hoc** – ‘making it work somehow’ attitude
  - **Fundamental problem:** *limited composability and compositionality in heterogeneous systems lead to lack of constructivity in system design*

# Scientific Challenge: Foundations for Correct-by-Construction Design

**Goal: extend the limits of “correct-by-construction” design:**

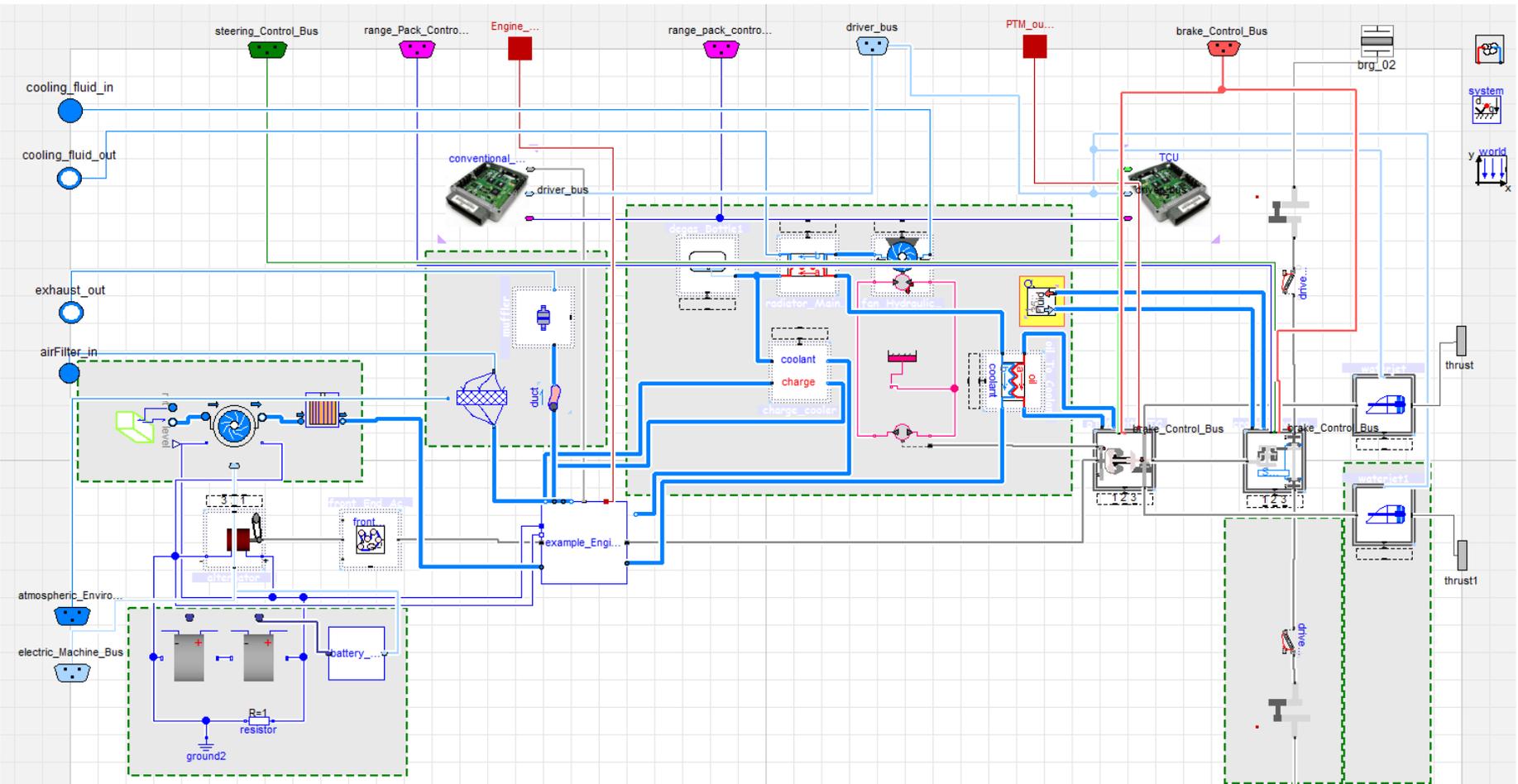
- **in *broad sense*: model- based design process that leads to manufacturable CPS products with desired properties**
- in *narrow sense*: use architectures that guarantee certain properties

**Three major challenges in CPS to advance correct-by construction:**

1. Multi-modeling with abstractions for modeling cross-domain interactions
2. Composition in heterogeneous domains
3. Validation and Verification



# Drivetrain Model in Modelica

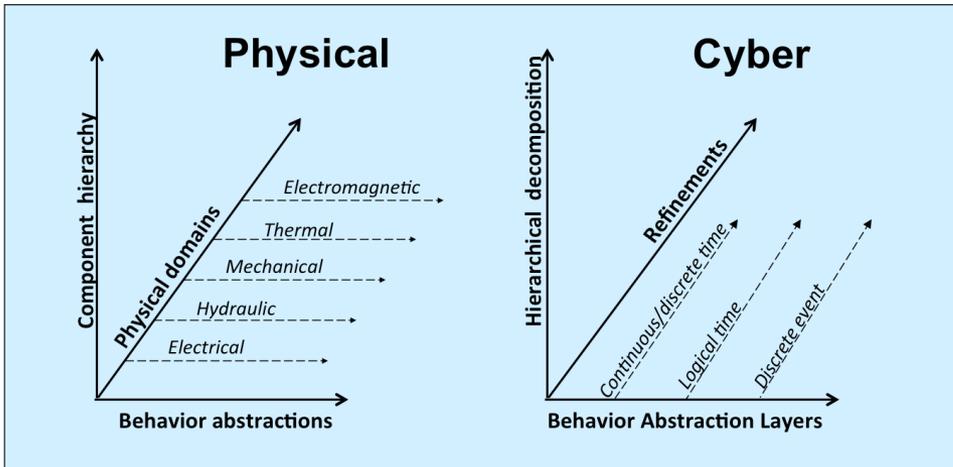


**Components:** Engine, Drive Shaft, Transmission, Final Drive, Air filter, Fuel-Pump, Fuel-Tank, Batteries, Alternator, Coolant System, Hydraulic Reservoir, Hydraulic Pump, ECU, TCU,

...



# Heterogeneity, Complexity, and Scale



## Imperative Code

```

block Sigmoid_Between_Limits
  "y = sigmoid(u) with y very close to 0 at u = u_low and very close to 1 at u = u_high"
  extends Modelica.Blocks.Interfaces.SISO;
  import C2M2L.Ext.Math.Logistic;
  //import C2M2L.Ext.Math.der_Logistic;
  parameter Real u_low = 0 "Value of u at which y should be very close to 0";
  parameter Real u_high = 1 "Value of u at which y should be very close to 1";
  parameter Real sharpness = 10
  "Higher values give a sharper transition and less deviation at u=u_low and u=u_high. Default 10 gives 0.7% deviation.";
  protected
    parameter Real mid_point = (u_high + u_low) * 0.5;
    parameter Real scale = u_high - u_low;
  equation
    assert(u_high > u_low, "Sigmoid_Between_Limits: u_high must be greater than u_low");
    y = Logistic(sharpness * (u - mid_point) / scale);
  end Sigmoid_Between_Limits;

```

### Original Model

Number of components: 1535 (MSL)  
 Variables: 14608  
 Constants: 323 (321 scalars)  
 Parameters: 5131 (15576 scalars)  
 Unknowns: 9154 (9911 scalars)  
 Differentiated variables: 341 scalars  
 Equations: 8449  
 Nontrivial : 7275

### Simplified Model

Constants: 4438 scalars  
 Free parameters: 7437 scalars  
 Parameter depending: 17496 scalars  
**Continuous-time states: 103 scalars**  
 Time-varying variables: 2476 scalars  
 Alias variables: 5249 scalars  
 Assumed default initial conditions: 78  
 Number of mixed real/discrete systems of equations: 15  
 Sizes of linear systems of equations: {31, 22, ...}  
 Sizes after manipulation of the linear systems: {22, 12, ...}  
 Sizes of nonlinear systems of equations: {70, 40, ...}  
 Sizes after manipulation of the nonlinear systems: {7, 6, ...}



# System Requirements



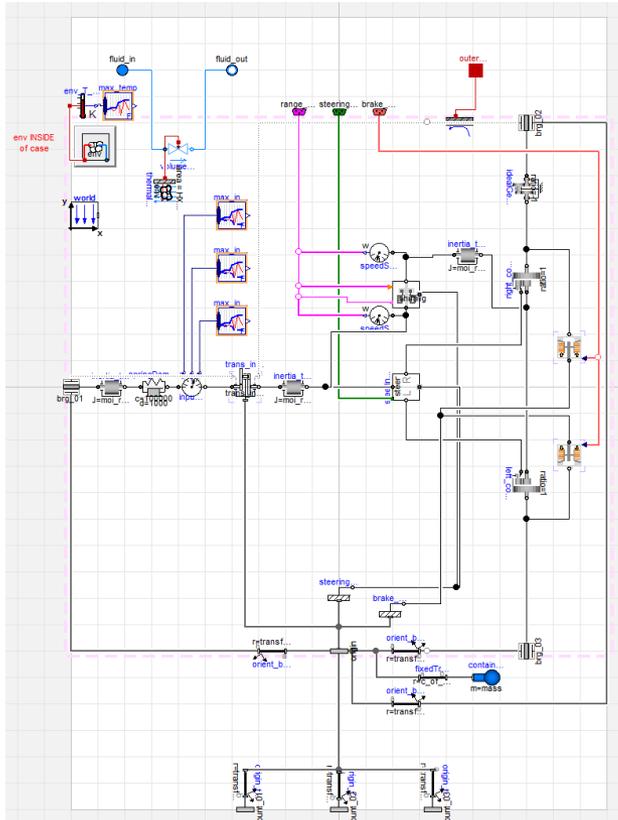
- Payload, towing load effects to dynamic behavior
- Operation in various environmental conditions
  - Altitude, temperature, ...
- Mobility
  - Operation profile, acceleration, speed, lateral stability, ...
- Brakes
  - Stopping distance, stopping time, ...
- Grade and slope operations
- Operational range
- Electronic stability control, suspension, transmission, tires, power generation and management
  - Dynamic behavior, time to raise lower adjustable height transmission, ...



The vehicle shall meet **all performance targets for all load conditions** without exceeding component manufacturers limits [JLTV FoV]



# Example: Transmission Limits



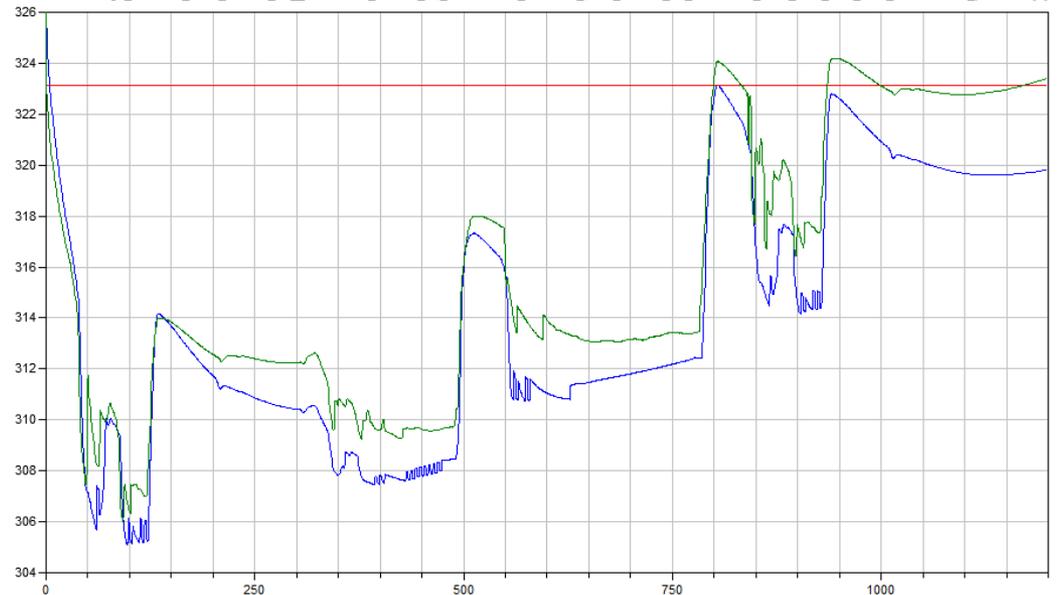
## SPECIFICATIONS

Gross Input Power — kW (hp)	895 (1200)
	1118 (1500)
Gross Input Torque — N•m (lb-ft)	
1200 hp	5484 (4045)
1500 hp	5995 (4422)
Maximum Turbine Torque —	
lb-ft (N•m)	7442 (5489)
Rated Input Speed — rpm	2100
Maximum Input Speed — rpm	2200
Weight Dry (approx) — kg (lb)	
Transmission	1694 (3735)
Engine/Transmission Coupling	108 (238)
Gears	
Type	Straight spur planetary
Forward/Reverse	7F/0R
Transmission Speed Ratios	

## OIL SYSTEM

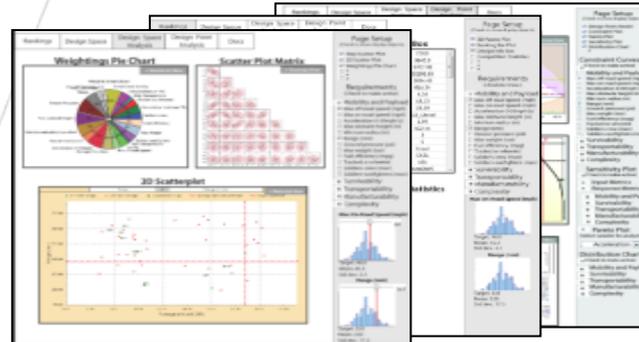
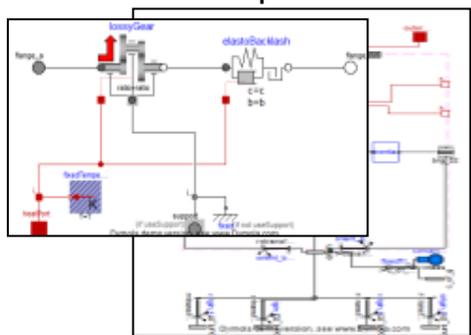
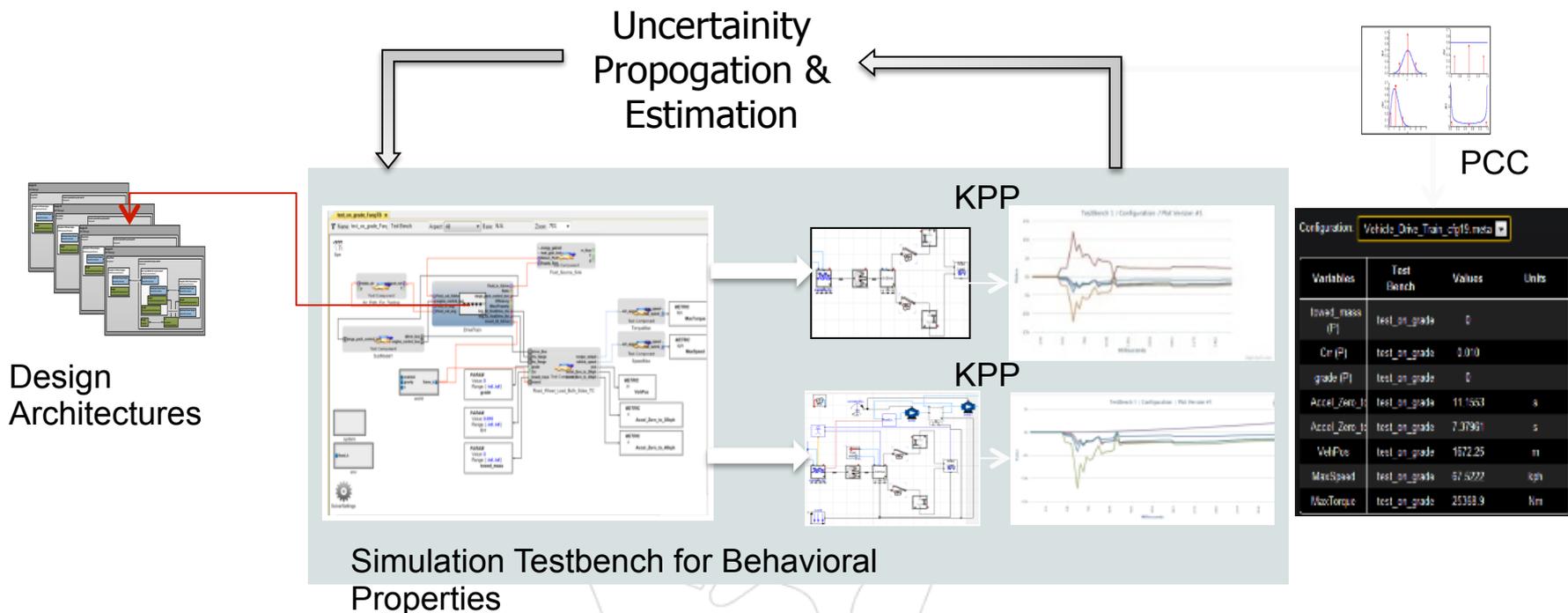
Cat Transmission/Drive Train Oil-4 (TDTO) or equivalent  
 Oil temperatures  
 Continuous — 90°C (195°F)  
 Maximum operating — 99°C (210°F)  
 Hydraulic fill capacity — approx 114 L (30 gal) subject to cooler size, lines, and installation — initial fill may be greater  
 Filter type — 6 micron synthetic, cartridge remote mount

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 — design\_v1.cross\_Drive\_without\_TC\_CrossDrive\_without\_TC\_PerkinsX300\_12.Cross\_Drive\_without\_TC\_Ext.volume\_Flow\_Rate\_With\_Heat\_Addition.heat\_port.T // 2 [K]





# Simulation-based Analysis

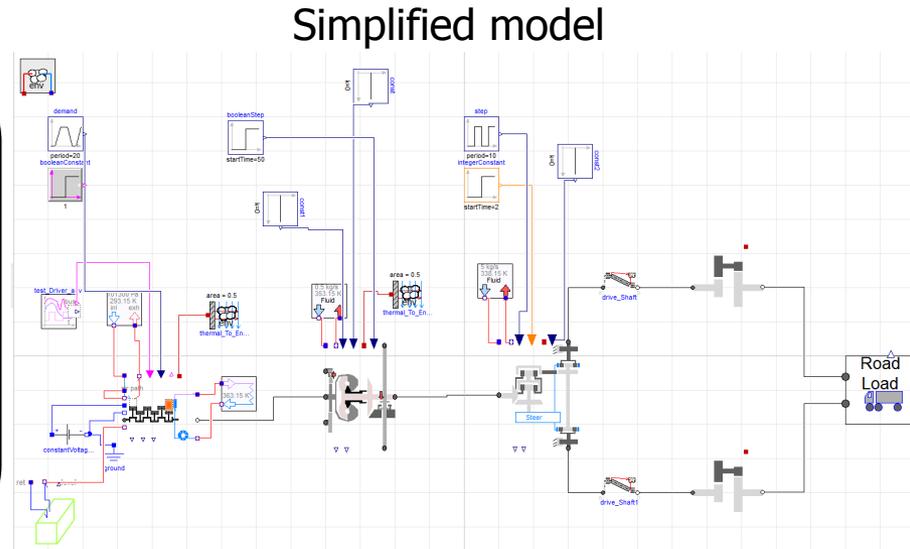
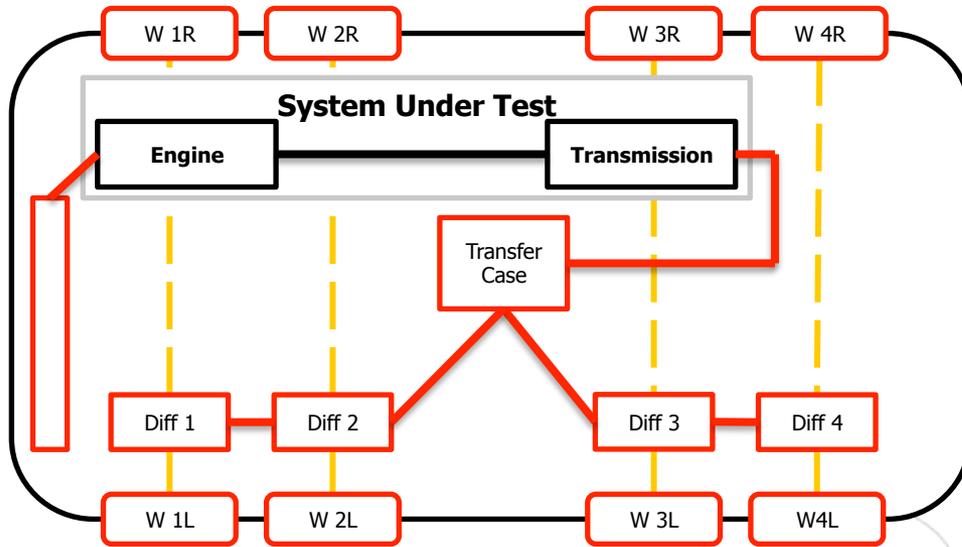


*Multiple Fidelity Behavior Models*

*Multiple Physics Domains*



# Hybrid System Verification



- Problem 1: Determine the range of grades where gears provably do not chatter
- Problem 2: Determine the range for input that guarantees that the engine RPM is bounded
  - Inputs: throttle position and grade of the road





- Complexity and scalability
- Models appropriate for formal verification
- Translation of design requirements to formal verification requirements
- Software tools and integration with modeling and simulation tools
- Usability
- Integration in the design flow: Design space exploration, guidance to the designer



- Model-based design relies on the **credibility of predictive modeling**
  - Computational models are used to predict system behavior which is not tested experimentally
- Verified properties are properties of the models
  - Dealing with modeling uncertainty is essential
    - Probabilistic uncertainty
    - Epistemic uncertainty
- Model validation is crucial
  - Fidelity-to-data
  - Robustness-to-uncertainty (and lack of knowledge)
  - Prediction confidence